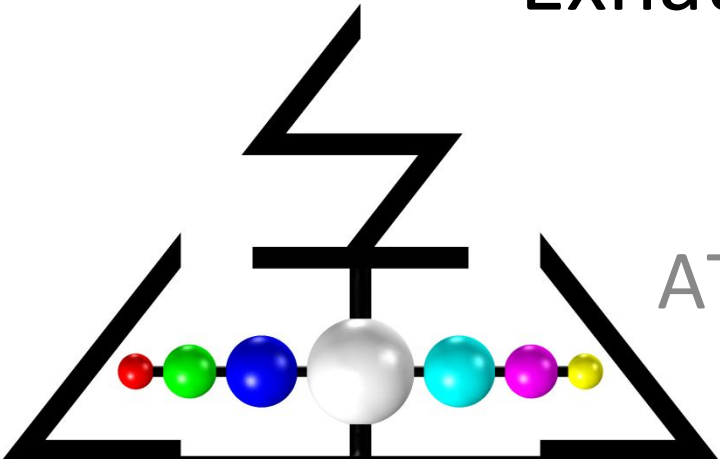


Multiphysics Modeling and Optimization of Automotive Heat Exchanger for Exhaust Waste Heat Recovery

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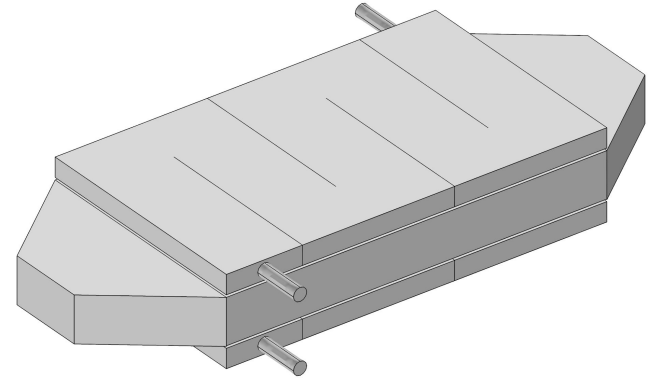
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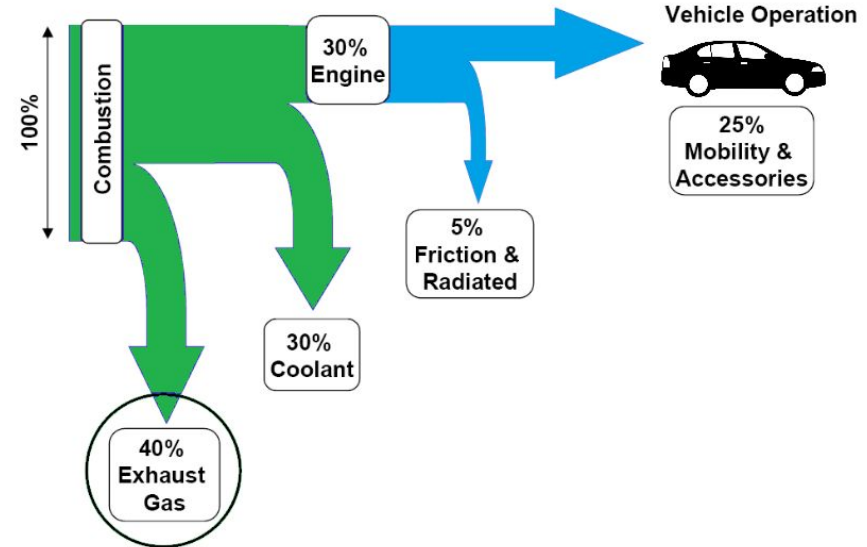
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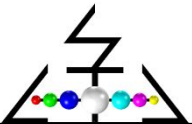


Introduction & Objectives

- Internal combustion engines are mechanical devices that produce work from fossil fuel combustion.
- Approximately 70% energy is wastage while remaining 30% used for vehicle operation.
- Huge loss in terms of fuel efficiency.
- Exhaust heat recovery technologies are used improve fuel efficiency indirectly by converting waste heat into useful energy.
- In this work a heat exchanger model is designed for efficient heat recovery from exhaust gas.

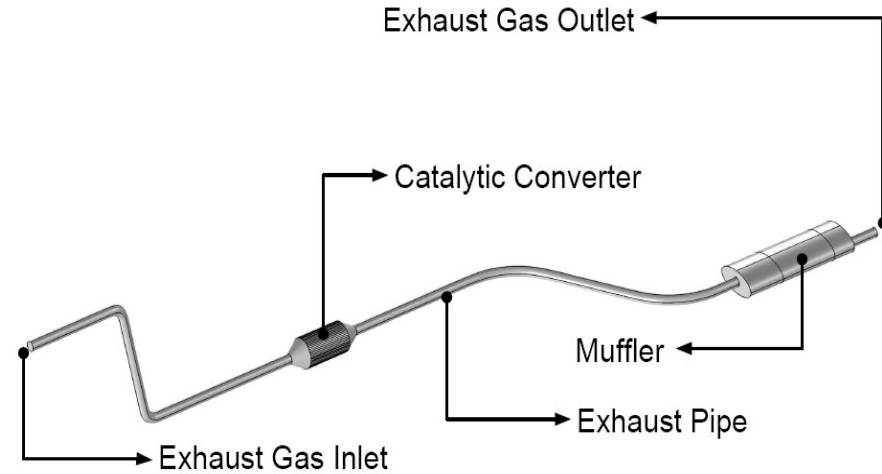


(Fig 1: Automotive Energy Flow, Sankey Diagram)

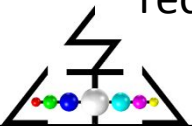


Automotive Exhaust System

- It guides the toxic gases from engine to the atmosphere.
- The responsibility and the job of an exhaust system is grown.
- Modern Exhaust Systems plays vital role to increase fuel efficiency
- Major Components
 - Exhaust Manifold
 - Manifold Pipe and Connector
 - Exhaust Pipe and Elbows
 - Catalytic Converter
 - Muffler
- Complex and challenging for exhaust heat recovery.

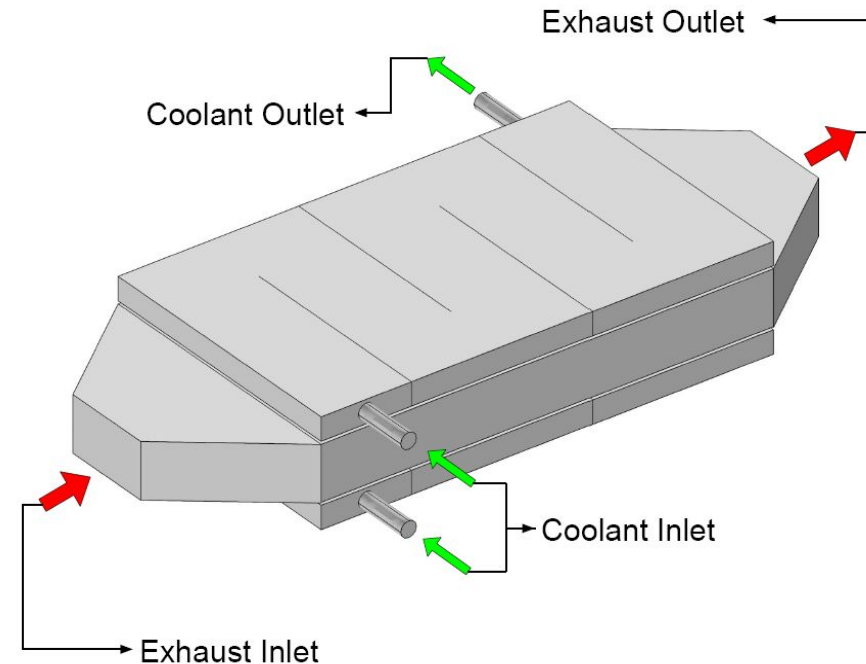


(Fig 2: Automotive Exhaust System)

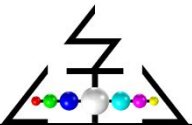


Heat Exchanger

- It is a mechanical device used to transfer heat between two fluids with different temperatures.
- Fluids are separated by metal walls to prevent direct contact and mixing.
- Heat exchanger design in this experiment for automotive exhaust heat recovery is given in figure 3.
- The heat exchanger is designed suitable for Thermoelectric application where proper temperature differential is needed for electric power generation.

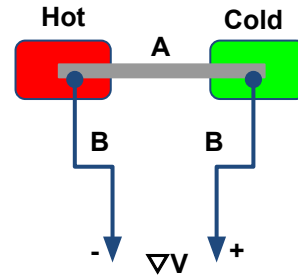


(Fig 3: Heat Exchanger)



Thermoelectric Generator (TEG)

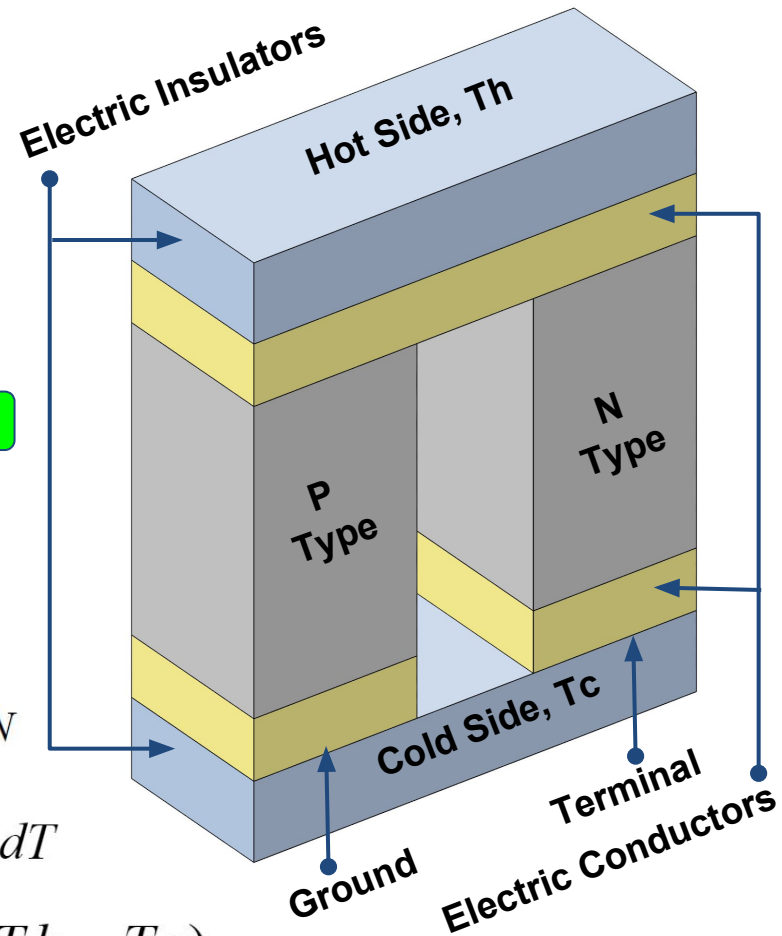
- Energy harvesting device.
- Convert temperature difference ($T_h - T_c$) into electrical energy (Voltage).
- Thermoelectricity.
 - Seebeck effect.
- Thermoelectric materials
 - Bismuth Telluride (Bi_2Te_3)
 - Lead telluride (PbTe)
 - Silicon Germanium (SiGe)
 - Skutterudite (CoAs_3)
- Seebeck Coefficient (α)
- Thermal Conductivity (k)
- Figure of Merit (ZT)



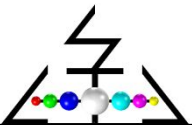
$$\alpha = \alpha_P - \alpha_N$$

$$V_{OC} = \int_{T_c}^{T_h} \alpha(T) dT$$

$$\nabla V \text{ or } V_{OC} = \alpha(T_h - T_c)$$

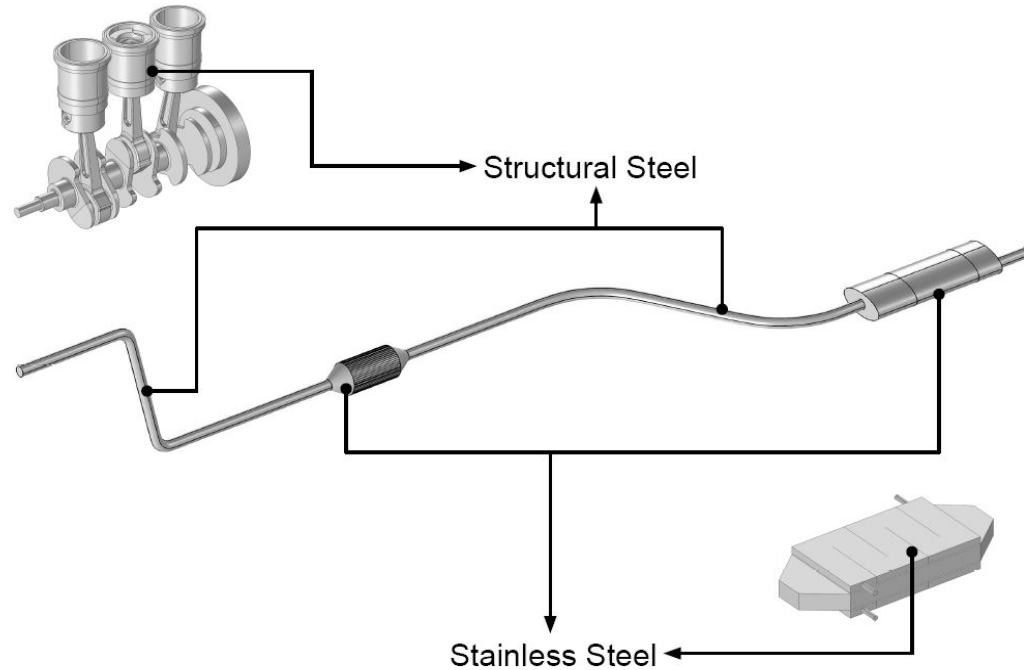


(Fig 4: TEG Module)



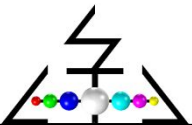
Material Properties

- The material properties such as Density, Thermal Conductivity, Heat capacity for engine and exhaust system components are given in the below table
- The material definition for engine and exhaust system is shown in figure 6.
- Exhaust pipes are defined as structural steel. Catalytic converter body and Muffler bodies are defined as stainless steel. The Catalyst elements in catalytic converter are defined as platinum.
- The fluid domains for the couple physics are defined as carbon dioxide for exhaust gas and water as coolant.



Materials	Cp (J/kgK)	K (W/mK)	ρ (kg/m ³)
Structural Steel	445	44.5	7850
Stainless Steel	530	17	8.07E-6
Platinum	133	71.6	21450

(Fig 6: Exhaust System Material Properties)



Comsol Simulation

Governing Equations

Turbulent Fluid Flow

$$\rho(u \cdot \nabla)u =$$

$$\nabla \cdot [-pI + (\mu + \mu_T)(\nabla u + (\nabla u)^T)] - \frac{2}{3}(\mu + \mu_T)(\nabla \cdot u)I + F$$

$$\nabla \cdot (\rho u) = 0$$

$$\rho(u \cdot \nabla)\kappa = \nabla \cdot \left[\left(\mu + \frac{\mu_T}{\sigma_\kappa} \right) \nabla \kappa \right] + P_\kappa - \rho \epsilon$$

$$\rho(u \cdot \nabla)\epsilon = \nabla \cdot \left[\left(\mu + \frac{\mu_T}{\sigma_\epsilon} \right) \nabla \epsilon \right] + C_{\epsilon 1} \frac{\epsilon}{\kappa} P_\kappa - C_{\epsilon 2} \rho \frac{\epsilon^2}{\kappa}$$

$$\epsilon = ep$$

$$\mu_T = \rho C_\mu \frac{\kappa^2}{\epsilon}$$

$$P_\kappa = \mu_T \left[\nabla u : (\nabla u + (\nabla u)^T) - \frac{2}{3}(\nabla \cdot u)^2 \right] - \frac{2}{3} \rho \kappa \nabla \cdot u$$

Heat Transfer in Solid

$$\rho C_p u \cdot \nabla T + \nabla \cdot q = Q + Q_{ted}, \quad q = -k \nabla T$$

Heat Transfer in Fluid

$$\rho C_p u \cdot \nabla T + \nabla \cdot q = Q + Q_p + Q_{vd}, \quad q = -k \nabla T$$

ρ = Density (kg/m³)

C_p = Specific heat (J/kg.K)

Q = Heat source (W/m³)

Q_{ted} = Thermoelastic effects

Q_p = Work done by pressure changes

Q_{vd} = Viscous dissipation in fluid

q = Heat flux by conduction (W/m²)

k = Thermal conductivity (W/m.K) (Heat Transfer)

k = Turbulent kinetic energy (Turbulent Flow)

T = Absolute temperature (K)

p = Pressure (Pa)

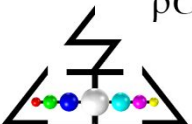
u = Velocity Vector (m/s)

μ = Dynamic Viscosity (Pa.s)

F = Volume force vector (N/m³)

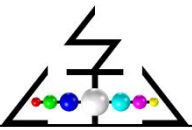
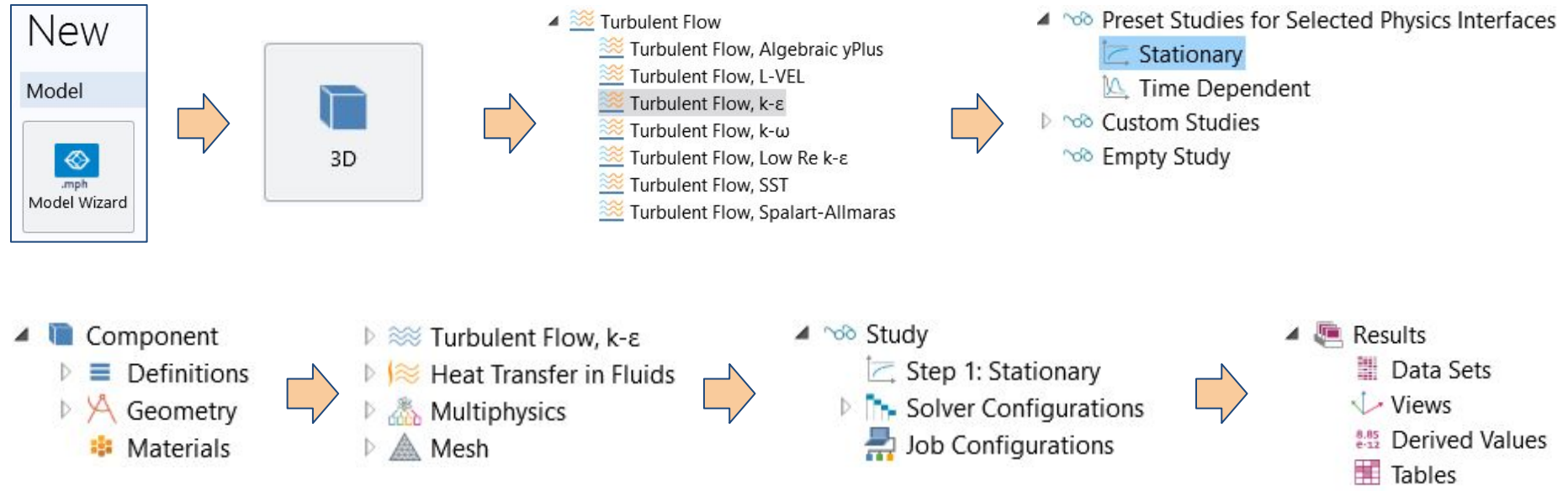
V = Velocity (m/s)

ep = Turbulent dissipation rate



Simulation in Comsol

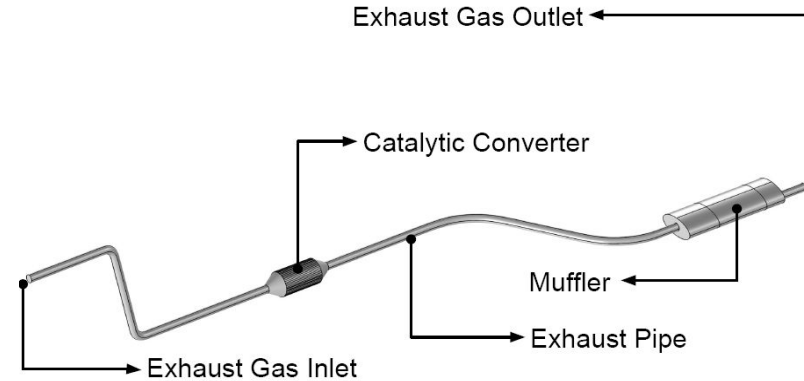
Steps



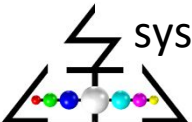
Automotive Exhaust System

Comsol Model

- A multiphysics cae model for the automotive exhaust system is developed using COMSOL.
- The computational model of the exhaust system is developed as shown in figure 1.8.
- The turbulent fluid flow, heat transfer in fluid and heat transfer in solid interfaces are coupled numerically.
- The exhaust gas temperature predicted from the fuel combustion (400^{C}) is taken as inlet exhaust temperature.
- A fully coupled, direct, stationery study is implemented to the automotive exhaust system.



(Fig 7: Exhaust System, Comsol Model)

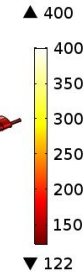


Simulation Results

Exhaust System, Contour Plots

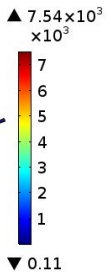
- The temperature distribution, exhaust pressure and fluid velocity magnitudes are plotted graphically as shown in contour plots.
- The maximum temperature occurrence is predicted to be nearer the engine while lesser at the tailpipe
- From the temperature distribution results in exhaust system the desired position for heat exchanger is predicted to be in between catalytic converter and muffler.

Surface: Temperature (degC)



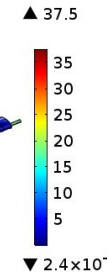
Temperature (degC)

Surface: Conductive heat flux magnitude (W/m²)



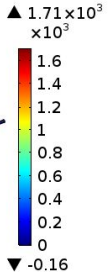
Conductive Heat Flux (W/m²)

Streamline: Velocity field (m/s)

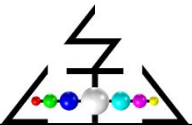


Fluid Velocity (m/s)

Surface: Pressure (Pa)



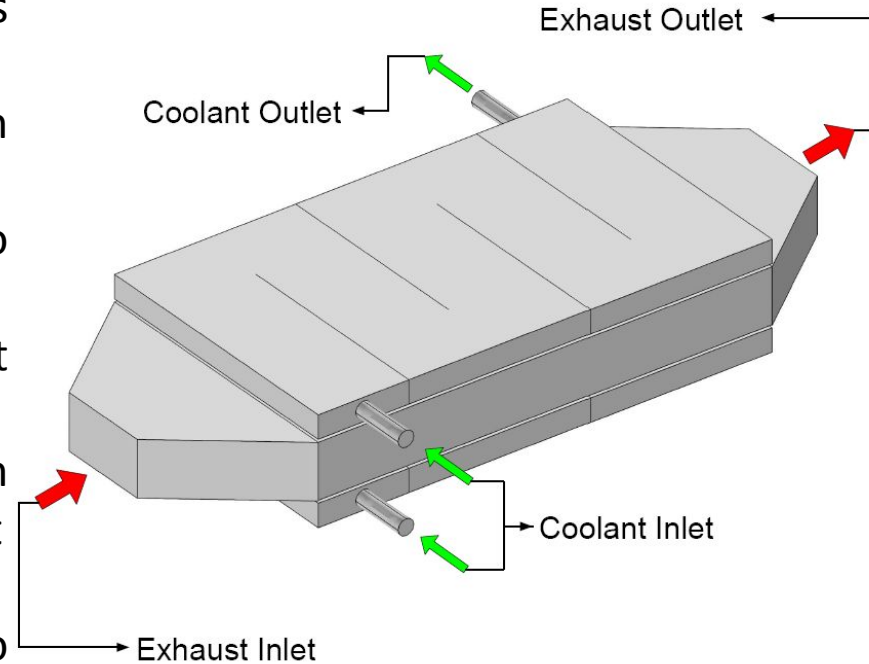
Fluid Pressure (Pa)



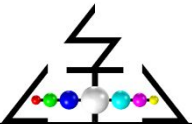
Heat Exchanger Simulation

Comsol Model

- An optimized heat exchanger model is designed in COMSOL.
- It will be attached to exhaust system in between catalytic converter and muffler.
- Two coolant chamber attached opposites to each other as shown in figure 8.
- Numerical boundary definition for heat exchanger model is defined as in figure 8.
- Exhaust gas temperature predicted from exhaust system (250°C) is taken as exhaust inlet temperature and coolant as 13°C .
- Corrugated chambers are designed to recover more heat from the exhaust gas.



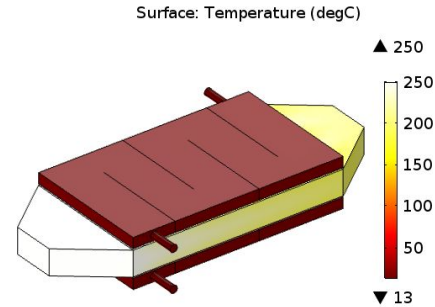
(Fig 8: Heat Exchanger, Comsol Model)



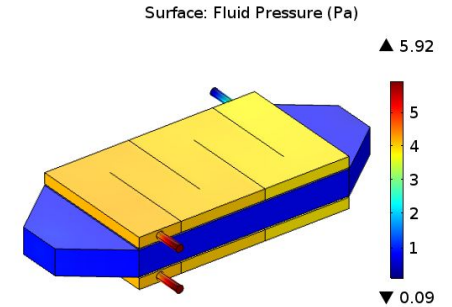
Simulation Results

Heat Exchanger, Contour Plots

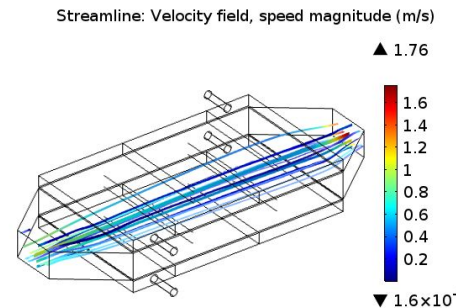
- Temperature distribution, exhaust pressure and fluid velocity magnitudes are plotted graphically as shown in contour plots.
- Maximum temperature occurrence is predicted to be at the entrance of the heat exchanger.
- Temperature distribution in the heat exchanger as shown is suitable for thermoelectric power generation.
- Thermoelectric modules can be attached safely in between exhaust chamber and coolant chamber to produce electric power.



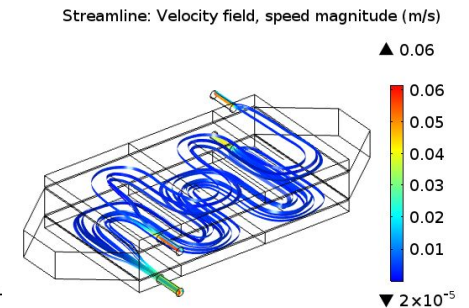
Temperature (degC)



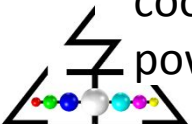
Fluid Pressure (Pa)



Exhaust Flow Velocity (m/s)



Coolant Flow Velocity (m/s)



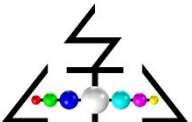
Conclusion and Future Work

- The waste heat produced during vehicle operation decreases fuel economy.
- In this paper a solution of waste heat recovery technology and method for better fuel economy is proposed by help of numerical simulation.
- The predicted temperature distribution in the exhaust system and heat exchanger shows potential in maximum exhaust waste heat recovery.
- Simulation results of heat exchanger found to be a suitable condition for thermoelectric power generation.
- The predicted simulation results can further be helpful in optimized design and placement of heat exchanger for maximum heat recovery.



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